

**Homework Assignment Number Eleven**  
**Assigned: Wednesday, April 7, 1999**  
**Due: Wednesday, April 14, 1999 BEFORE LECTURE STARTS.**

**Problem 1.** Geankoplis 4.9-2, page 323

**Solution:**

$$q = UA\Delta T_{lm} = \dot{m}_{\text{tube}} C_p (\bar{T}_{\text{tube}})(T_{\text{tube,out}} - T_{\text{tube,in}})$$

$$q = UA\Delta T_{lm} = \dot{m}_{\text{shell}} C_p (\bar{T}_{\text{shell}})(T_{\text{shell,out}} - T_{\text{shell,in}})$$

Oil is cooled on the tube-side of the exchanger.

$$\dot{m}_{\text{tube}} C_p (\bar{T}_{\text{tube}})(T_{\text{tube,out}} - T_{\text{tube,in}}) = 5.04 \cdot 2090 \cdot (344.3 - 366.5)$$

$$q = -233846 \text{ W}$$

Water is heated on the shell-side of the exchanger:

$$q = \dot{m}_{\text{shell}} C_p (\bar{T}_{\text{shell}})(T_{\text{shell,out}} - T_{\text{shell,in}})$$

$$q = 233846 \text{ W} = 2.02 \cdot 4184 (T_{\text{shell,out}} - 283.2)$$

$$T_{\text{shell,out}} = 310.9$$

Since this is a 1-2 exchanger, calculate Y and Z to obtain  $F_T$

$$Z = \frac{T_{\text{hot,in}} - T_{\text{hot,out}}}{T_{\text{cold,out}} - T_{\text{cold,in}}} = 0.801$$

$$Y = \frac{T_{\text{cold,out}} - T_{\text{cold,in}}}{T_{\text{hot,in}} - T_{\text{cold,in}}} = 0.333$$

For 1-2 shell and tube heat exchangers, compute the correction factor,  $F_T$ , for the log mean T difference from Figure 4.9-4(a) on page 270, Geankoplis.

$$F_T = 0.96$$

(Assume counter current flow.)

$$\Delta T_{lm} = \frac{(T_{\text{shell,in}} - T_{\text{tube,out}}) - (T_{\text{shell,out}} - T_{\text{tube,in}})}{\ln\left(\frac{(T_{\text{shell,in}} - T_{\text{tube,out}})}{(T_{\text{shell,out}} - T_{\text{tube,in}})}\right)} = 58.3\text{K}$$

$$\Delta T_{lm} = F_T \Delta T_{lm} = 0.96 \cdot 58.3 = 56.0K$$

$$q = 233846 \text{ W} = UA\Delta T_{lm} = 340 \cdot A \cdot 56.0$$

$$A = 12.3m^2$$

**Problem 2.** Geankoplis 4.9-3, page 323

**Solution:**

$$q = UA\Delta T_{lm} = \dot{m}_{\text{tube}} C_p (\bar{T}_{\text{tube}})(T_{\text{tube,out}} - T_{\text{tube,in}})$$

$$q = UA\Delta T_{lm} = \dot{m}_{\text{shell}} C_p (\bar{T}_{\text{shell}})(T_{\text{shell,out}} - T_{\text{shell,in}})$$

Water is heated on the tube-side of the exchanger:

$$\dot{m}_{\text{tube}} C_p (\bar{T}_{\text{tube}})(T_{\text{tube,out}} - T_{\text{tube,in}}) = 1.13 \cdot 4184 \cdot (358 - 318)$$

$$q = 189117 \text{ W}$$

Oil is cooled on the shell-side of the exchanger.

$$q = \dot{m}_{\text{shell}} C_p (\bar{T}_{\text{shell}})(T_{\text{shell,out}} - T_{\text{shell,in}})$$

$$q = 189117 \text{ W} = \dot{m}_{\text{shell}} \cdot 1950(358 - 393)$$

$$\dot{m}_{\text{shell}} = 2.77 \frac{\text{kg}}{\text{s}}$$

Since this is a 1-2 exchanger, calculate Y and Z to obtain  $F_T$

$$Z = \frac{T_{\text{hot,in}} - T_{\text{hot,out}}}{T_{\text{cold,out}} - T_{\text{cold,in}}} = 0.875$$

$$Y = \frac{T_{\text{cold,out}} - T_{\text{cold,in}}}{T_{\text{hot,in}} - T_{\text{cold,in}}} = 0.533$$

For 1-2 shell and tube heat exchangers, compute the correction factor,  $F_T$ , for the log mean T difference from Figure 4.9-4(a) on page 270, Geankoplis.

$$F_T = 0.83$$

(Assume counter current flow.)

$$\Delta T_{lm} = \frac{(T_{shell,in} - T_{tube,out}) - (T_{shell,out} - T_{tube,in})}{\ln\left(\frac{(T_{shell,in} - T_{tube,out})}{(T_{shell,out} - T_{tube,in})}\right)} = 37.4K$$

$$\Delta T_{lm} = F_T \Delta T_{lm} = 0.83 \cdot 37.4 = 31.1K$$

$$q = 189117W = UA\Delta T_{lm} = 300 \cdot A \cdot 31.1$$

$$A = 20.3m^2$$

### Problem 3.

Redo parts (b) and (c) from Problem 3, Homework 10, using the Matlab code given at <http://clausius.engr.utk.edu/che240/text/codes.html>

(b) Use your code to solve the following problem:

We will be using cold water to cool hot water in a 1-1 shell and tube heat exchanger. The hot water is inside the tubes. The cold water is in the shell. The flow is counter-current.

The cold water enters the shell at  $T_{shell,in} = 250K$ . The hot water enters the tubes at

$T_{tube,in} = 400K$ . The flow-rate of the hot stream is  $\dot{m}_{tube} = 0.10 \frac{kg}{s}$  and the flow-rate of the cold stream is  $\dot{m}_{shell} = 0.20 \frac{kg}{s}$ .

The heat exchanger is made of stainless steel tubes of outside diameter 1 inch, BWG 10. The length of the tubes and the length of the shell are  $L = 4.0m$ ; the number of tubes,  $N_{tube} = 24$ ; the diameter of the shell,  $D_{shell} = 0.15m$ . Use fouling factors for city water.

Determine the six quantities specified in part (a)

(c) Repeat Part (b) with cocurrent flow.

### Solution:

(a) I used the code, hxchnger\_v2.m available on the course website.

(b) I iterated until both outlet temperatures were constant within 0.00001 Kelvin.

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» hxchger_v2
initial guess: Ttubeout = 370.000000 Tshellout = 281.500000
at j = 1, i = 25: Ttubeout = 298.569505 Tshellout = 340.750000
at j = 2, i = 25: Ttubeout = 311.014789 Tshellout = 311.125000
...
at j = 23, i = 24: Ttubeout = 301.612188 Tshellout = 298.940543
at j = 24, i = 24: Ttubeout = 301.612182 Tshellout = 298.940536

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Tube-side Temp (K) inlet: 400.000000, outlet 301.612182, avg 350.806088  
 Shell-side Temp (K) inlet: 250.000000, outlet 298.940536, avg 274.470271  
 Wall Temp (K): Tube-side = 300.771756, Shell-side = 295.300084, avg = 298.035920  
 Heat Transfer coefficients shell film, heat film, overall (W/m<sup>2</sup>/K):  
 hshell = 259.205906 htube = 147.361123 Uo = 73.371215  
 Uo = 73.371215 Ui = 100.195101  
 Heat Transferred (W): q = 41360.433298 q<sub>tube</sub> = 41360.434493 q<sub>shell</sub> = 41360.427860  
 Enthalpy Changes (W): tube = -41374.088821 shell = 41360.436210

**(c)** For cocurrent flow, I had to change the variable flow from 1 to 0.  
 Then I reran the code:

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» hxchger_v2
initial guess: Ttubeout = 370.000000 Tshellout = 281.500000
at j = 1, i = 25: Ttubeout = 306.997020 Tshellout = 340.750000
at j = 2, i = 25: Ttubeout = 343.078103 Tshellout = 311.125000
...
at j = 23, i = 25: Ttubeout = 312.982627 Tshellout = 293.403824
at j = 24, i = 25: Ttubeout = 312.982632 Tshellout = 293.403831
Tube-side Temp (K) inlet: 400.000000, outlet 312.982632, avg 356.491319
Shell-side Temp (K) inlet: 250.000000, outlet 293.403831, avg 271.701912
Wall Temp (K): Tube-side = 314.655192, Shell-side = 291.497858, avg = 303.076525
Heat Transfer coefficients shell film, heat film, overall (W/m2/K):
hshell = 242.314115 htube = 156.575664 Uo = 74.890121
Uo = 74.890121 Ui = 102.269306
Heat Transferred (W): q = 36745.975914 qtube = 36745.977922 qshell = 36745.982630
Enthalpy Changes (W): tube = -36667.588518 shell = 36745.966614
  
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Comparing with (b), we see that 11.2% less heat is transferred in the cocurrent case.