

Homework Assignment Number Two Solutions
Assigned: Wednesday, January 20, 1999
Due: Wednesday, January 27, 1999 BEGINNING OF CLASS.

Problem 1. Geankoplis, problem 2.3-2, page 105

$$\frac{\partial \Gamma}{\partial t} = \delta \frac{\partial^2 \Gamma}{\partial z^2} + R \quad \text{eqn. (2.3-11), page 42.}$$

No accumulation, no generation.

$$0 = \frac{\partial^2 \Gamma}{\partial z^2} + 0$$

Now, the equation is no longer a PDE but an ODE.

$$\frac{\partial^2 \Gamma}{\partial z^2} = \frac{d^2 \Gamma}{dz^2} = \frac{d\left(\frac{d\Gamma}{dz}\right)}{dz} = 0$$

Integrate out second derivative.

$$\left(\frac{d\Gamma}{dz}\right)_{z=z_2} - \left(\frac{d\Gamma}{dz}\right)_{z=z_1} = \int_{z=z_1}^{z=z_2} 0 \cdot dz$$

The integral of zero yields some constant to be determined from the boundary conditions.

$$\left(\frac{d\Gamma}{dz}\right)_{z=z_2} - \left(\frac{d\Gamma}{dz}\right)_{z=z_1} = 0$$

Integrate out first derivative.

$$\Gamma(z'=z_2) - \Gamma(z'=z) = \int_{z'=z}^{z'=z_2} \left(\frac{d\Gamma}{dz}\right)_{z=z_1} dz'$$

$$\Gamma(z' = z_2) - \Gamma(z' = z) = \left(\left(\frac{d\Gamma}{dz}\right)_{z=z_1}\right)(z_2 - z)$$

$$\Gamma(z) = \Gamma_2 - \left(\left(\frac{d\Gamma}{dz} \right) \Big|_{z=z_1} \right) (z_2 - z)$$

at z_1 ,

$$\Gamma(z_1) = \Gamma_1 = \Gamma_2 - \left(\left(\frac{d\Gamma}{dz} \right) \Big|_{z=z_1} \right) (z_2 - z_1)$$

so, by rearranging:

$$\left(\frac{d\Gamma}{dz} \right) \Big|_{z=z_1} =$$

and substituting this result back into the equation for gamma:

$$\Gamma(z) = \Gamma_2 - \left(\frac{\Gamma_2 - \Gamma_1}{z_2 - z_1} \right) (z_2 - z)$$

This equation describes a straight line between the boundary points.

Problem 2. Geankoplis, problem 2.4-2, page 105

Using figure 2.4-1. $v=0.4\text{m/s}$. Fluid is water. $T= 24 \text{ C}$.

$$(a) \text{ Using equation (2.4-5), we have } \tau_{yz} = -\mu \frac{v_z(y_2) - v_z(y_1)}{y_2 - y_1}$$

$$\tau_{yz} = -\left(.9142 \cdot 10^{-3} \right) \frac{0.4\text{m/s}}{\Delta y} = -0.3\text{N/m}^2$$

$$\Delta y = \left(.9142 \cdot 10^{-3} \right) \frac{0.4\text{m/s}}{.3\text{N/m}^2} = 1.2 \cdot 10^{-3}\text{m}$$

$$\text{shear rate} = \frac{v_z(y_2) - v_z(y_1)}{y_2 - y_1} = 333.33 \text{ sec}^{-1}$$

(b) oil

$$\tau_{yz} = -(0.02) \frac{0.4}{.0012} = -6.6667 \frac{\text{N}}{\text{m}^2}$$

$$\text{shear rate} = \frac{v_z(y_2) - v_z(y_1)}{y_2 - y_1} = 333.33 \text{ sec}^{-1}$$

Problem 3. Geankoplis, problem 2.5-1, page 105

$$(a) \quad A = \left(\frac{\pi D^2}{4} \right) = \frac{\pi}{4} \left(63.5 \text{mm} \cdot \frac{1 \text{m}}{1000 \text{mm}} \right)^2 = 0.00317 \text{m}^2$$

$$\dot{m} = \rho A v$$

$$v = \frac{\dot{m}}{\rho A} = \frac{0.605 \frac{\text{kg}}{\text{s}}}{1030 \frac{\text{kg}}{\text{m}^3} \cdot 0.00317 \text{m}^2} = 0.185 \frac{\text{m}}{\text{s}}$$

$$N_{\text{Re}} = \frac{D v \rho}{\mu} = \frac{\left(63.5 \text{mm} \cdot \frac{1 \text{m}}{1000 \text{mm}} \right) \left(0.185 \frac{\text{m}}{\text{s}} \right) \left(1030 \frac{\text{kg}}{\text{m}^3} \right)}{2.12 \text{cp} \cdot \frac{10^{-3} \text{kg}}{\text{m} \cdot \text{s}}} = 5710$$

TURBULENT

$$(b) \quad N_{\text{Re}} = \frac{D v \rho}{\mu} = \frac{(0.0635 \text{m}) \left(v \frac{\text{m}}{\text{s}} \right) \left(1030 \frac{\text{kg}}{\text{m}^3} \right)}{0.00212 \frac{\text{kg}}{\text{m} \cdot \text{s}}} = 2100$$

$$v = 0.0681 \frac{\text{m}}{\text{s}}$$

$$q = A v = 0.00317 \text{m}^2 \cdot \left(0.0681 \frac{\text{m}}{\text{s}} \right) = 2.16 \cdot 10^{-4} \frac{\text{m}^3}{\text{s}}$$

Problem 4. Geankoplis, problem 2.6-1, page 106

$$v_z = \frac{\rho g \delta^2}{2\mu} \left[1 - \left(\frac{x}{\delta} \right)^2 \right]$$

$$(a) \quad v_{z,\text{max}} = \frac{\rho g \delta^2}{2\mu} \text{ at } x = 0, \text{ by inspection, or:}$$

Take the derivative with respect to x and set it equal to zero. Solving this for the velocity, will give the extrema of the velocity.

$$(b) \bar{v} = \frac{\iint_A v dA}{\iint_A dA} = \frac{1}{A} \iint_A v dA$$

which for free flow down a vertical plate of width, W, becomes

$$\bar{v} = \frac{1}{W\delta} \int_0^\delta \int_0^W v_{\max} \left[1 - \left(\frac{x}{\delta} \right)^2 \right] dy dx$$

There is no y-dependence on the velocity (across the width of the plate) so that integrates out as a W and cancels with the W in the denominator.

$$\begin{aligned} \bar{v} &= \frac{W}{W\delta} \int_0^\delta v_{\max} \left[1 - \left(\frac{x}{\delta} \right)^2 \right] dx \\ \bar{v} &= \frac{v_{\max}}{\delta} \int_0^\delta \left[1 - \left(\frac{x}{\delta} \right)^2 \right] dx = \frac{v_{\max}}{\delta} \left[x - \frac{x^3}{3\delta^2} \right]_0^\delta = \frac{v_{\max}}{\delta} \left[\delta - \frac{\delta^3}{3\delta^2} \right] \\ \bar{v} &= \frac{v_{\max}}{\delta} \left[\delta - \frac{\delta}{3} \right] = \frac{2v_{\max}}{3\delta} \delta = \frac{2v_{\max}}{3} \end{aligned}$$

Problem 5. Geankoplis, problem 2.6-2, page 106

$$\dot{m}_{in} = \rho A v = \left(902 \frac{\text{kg}}{\text{m}^3} \right) (0.00433 \text{m}^2) \left(1.282 \frac{\text{m}}{\text{s}} \right) = 5.01 \frac{\text{kg}}{\text{s}}$$

$\dot{m}_{out} = \dot{m}_{in}$ by conservation of mass and the steady state assumption.

$$v_{out} = \frac{\dot{m}}{\rho A} = \frac{5.01 \frac{\text{kg}}{\text{s}}}{875 \frac{\text{kg}}{\text{m}^3} \cdot 0.00526 \text{m}^2} = 1.09 \frac{\text{m}}{\text{s}}$$

$$\text{massflux}_{in} = \frac{\dot{m}}{A} = \frac{5.01 \frac{\text{kg}}{\text{s}}}{0.00433 \text{m}^2} = 1156 \frac{\text{kg}}{\text{s} \cdot \text{m}^2}$$